



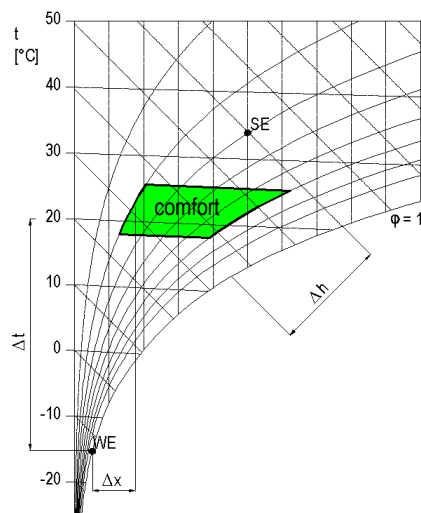
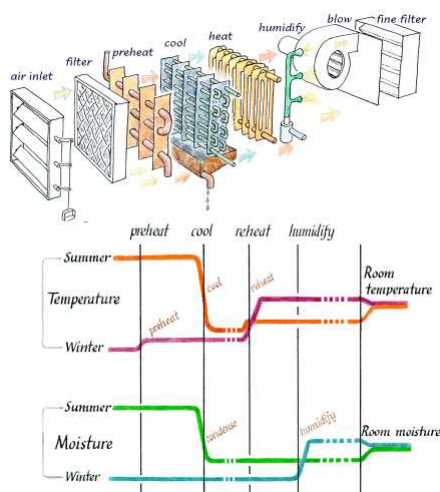
Ventilation

6 Heat Exchangers

Vladimír Zmrhal (room no. 814)

<http://users.fs.cvut.cz/~zmrhavl/index.htm>

Air-conditioning processes



■ Dimensioning of air-conditioning



Outdoor air conditions (Czech Republic)

WINTER

$$t_e = t_{e,loss} - 3 \text{ °C}, \varphi_e = 100 \% \quad (t_{e,loss} = -12, -15, -18 \text{ °C})$$

SUMMER

$$t_e = 32 \text{ °C}, h_e = 58 \text{ kJ/kg}$$

Indoor air conditions (for air-conditioned spaces)

Indoor air temperature $t_i = 18$ to 26 °C

Relative humidity $\varphi_i = 30$ to 70%

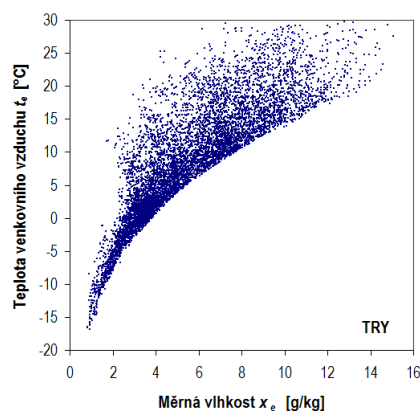
**Example 1: Find the
summer and winter
extreme in the
diagram**

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■ Air-conditioning processes



Typical climatic data (Czech Republic)



TRY – test reference
year

t_e – ambient air
temperature [°C]

x_e – humidity
ratio [g/kg]

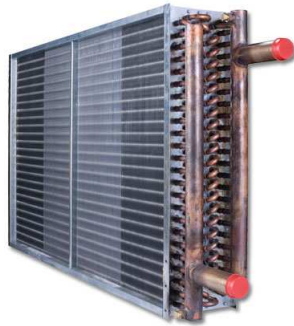
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Air heating



Heating coil

➤ water coil heat exchangers



electric heaters



direct heaters



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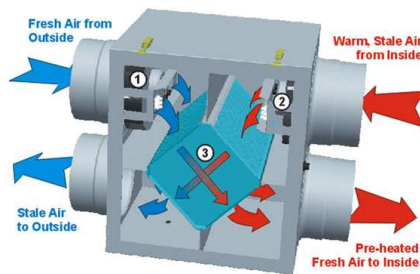
Air heating



Heat recovery

heat recuperation

heat regeneration



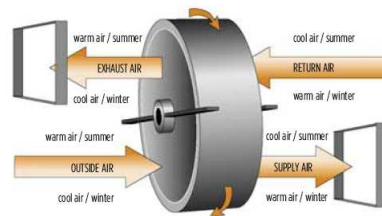
Example 2:

$$\Phi = 65\%$$

$$t_e = -15\text{ }^\circ\text{C}$$

$$t_i = 20\text{ }^\circ\text{C}$$

$$t_{sup} = ?$$



Recovered heat coefficient

$$\Phi = \frac{t_{e2} - t_{e1}}{t_{o1} - t_{e1}}$$

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Air heating



Heat flow rate

➤ heating coil capacity

$$Q_h = V\rho c(t_2 - t_1)$$

$$\Delta x = 0$$

$$\delta = \infty$$

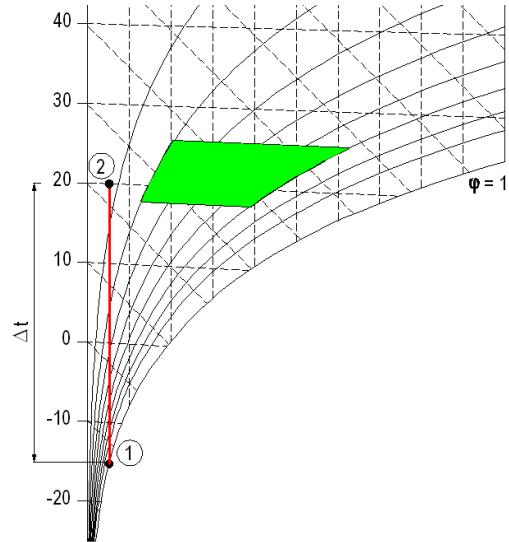
Example 2:

$$V = 10\,000 \text{ m}^3/\text{h}$$

$$t_1 = -15 \text{ }^\circ\text{C}$$

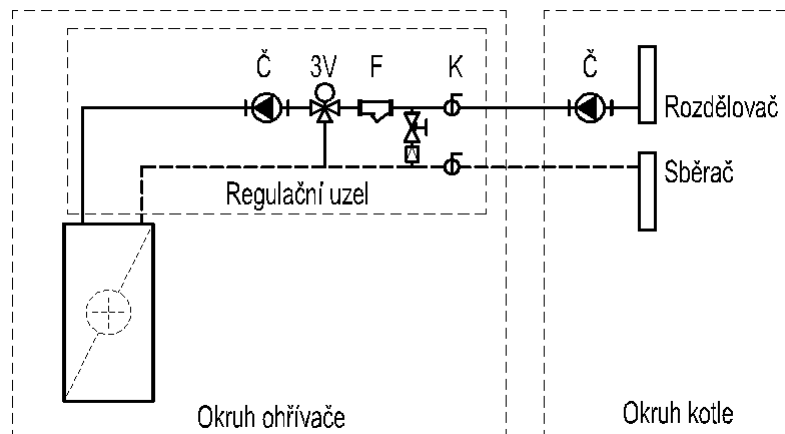
$$t_2 = 20 \text{ }^\circ\text{C}$$

$$Q_h = ?$$



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Air heating



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■ Air cooling



Sensible cooling - without condensation („dry cooling“)

➤ the temperature on a cooling surface is above or equal to the dew point temperature → $t_c > t_{dp}$

Latent cooling - dehumidifying moist air

➤ the temperature on a cold surface is lower than the dew point temperature → $t_c < t_{dp}$

Cold surface temperature

➤ water exchangers e.g. 6/12 °C

$$t_c = (t_{w1} + t_{w2})/2 = 9 \text{ °C}$$

➤ evaporators

$$t_c \approx 4 - 5 \text{ °C}$$

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■ Air cooling



Cooling coil capacity

➤ total heat flow rate through the cooling coil

$$Q_c = Q_{sen} + Q_{lat} = V\rho c(t_1 - t_2) + V\rho l(x_1 - x_2)$$

$$Q_c = V\rho(h_1 - h_2)$$

$$\Delta h = c\Delta t + l\Delta x$$

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Air cooling



Cooling without condensation

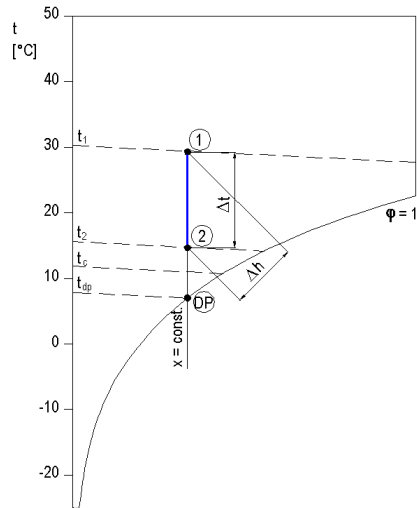
➤ $t_c > t_{dp}$

➤ cooling coil capacity

$$Q_{sen} = V\rho c(t_1 - t_2)$$

$$Q_{lat} = 0$$

$$Q_c = V\rho(h_1 - h_2) = \\ = V\rho c(t_1 - t_2)$$



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Air cooling

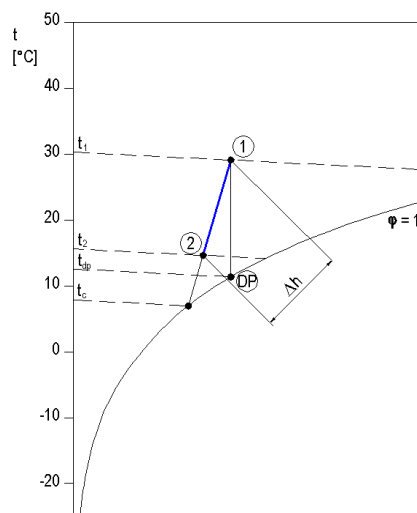


Dehumidifying moist air

➤ $t_c < t_{dp}$

➤ cooling coil capacity

$$Q_c = V\rho(h_1 - h_2)$$



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Air cooling



Cooling without condensation

Example 3a:

$V = 10\,000\text{ m}^3/\text{h}$

$t_1 = 32\text{ }^\circ\text{C}$

$h_1 = 58\text{ kJ/kg}$

$t_2 = 20\text{ }^\circ\text{C}$

$t_c = 16\text{ }^\circ\text{C}$

$Q_c = ?$

Cooling with dehumidifying

Example 3b:

$V = 10\,000\text{ m}^3/\text{h}$

$t_1 = 32\text{ }^\circ\text{C}$

$h_1 = 58\text{ kJ/kg}$

$t_2 = 20\text{ }^\circ\text{C}$

$t_{w1}/t_{w2} = 6/12\text{ }^\circ\text{C}$

$Q_c = ?$

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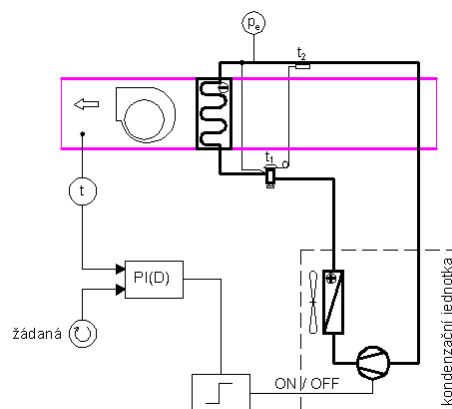
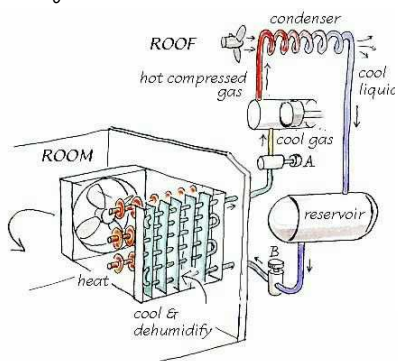
Air cooling



Direct air cooling

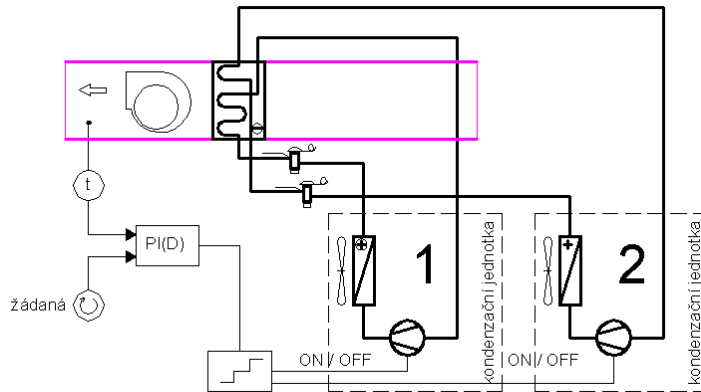
➤ cooling coil = evaporator

➤ $t_c = 4 - 5\text{ }^\circ\text{C}$



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■ Air cooling



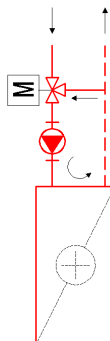
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■ Control of water exchangers



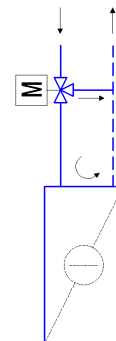
Supply temp. control

- mixing of the flow
- $V = \text{const.}$
- $t_w = \text{var.}$
- heating/cooling coils



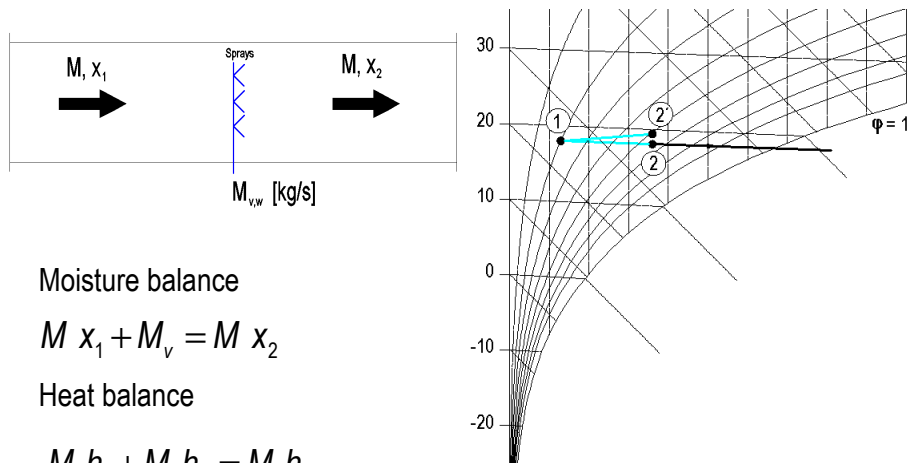
Heat mass flow control

- dividing of the flow
- $V = \text{var.}$
- $t_w = \text{const.}$
- cooling coils



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■ Humidifying (adding steam)



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■ Humidifying (adding steam)



Enthalpy of water vapor

$$h_v = l + c_v t = 2500 + 1.86 \cdot t \cong 2540 \quad [\text{kJ/kg}]$$

Direction of the process

$$\delta = \frac{\Delta h}{\Delta x} = \frac{h_2 - h_1}{x_2 - x_1} = h_v$$

$$\delta = h_v = 2.5 \quad [\text{kJ/g}]$$

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■ Humidifying (adding steam)



Enthalpy of water vapor – reality

$$l = 2500 \text{ kJ/kg} \quad \dots \text{ for } 0 \text{ }^\circ\text{C}$$

$$h_v = l + c_v t + c_w t = 2500 + 1.86 \cdot 100 + 4.2 \cdot 0 \cong 2680 \quad [\text{kJ/kg}]$$

$$l = 2257 \text{ kJ/kg} \quad \dots \text{ for } 100 \text{ }^\circ\text{C}$$

$$h_v = l + c_v t + c_w t = 2257 + 2.08 \cdot 0 + 4.2 \cdot 100 \cong 2680 \quad [\text{kJ/kg}]$$

$$\delta = h_v = 2.68 \quad [\text{kJ/g}]$$

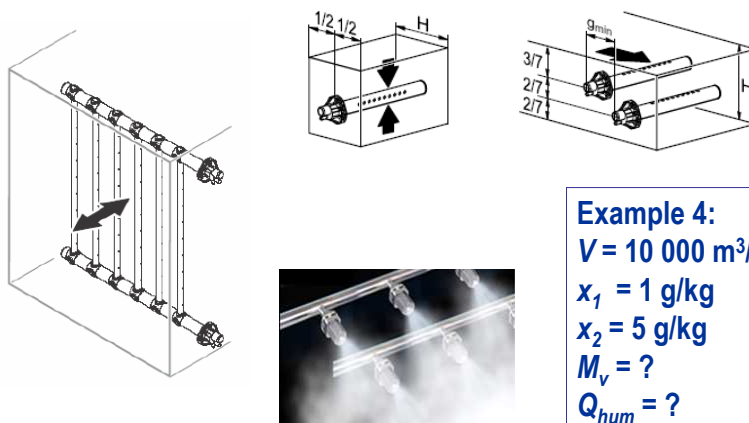
In reality the air is also heated !

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■ Humidifying (adding steam)



Distribution of steam (sprays)



Example 4:
 $V = 10\,000 \text{ m}^3/\text{h}$
 $x_1 = 1 \text{ g/kg}$
 $x_2 = 5 \text{ g/kg}$
 $M_v = ?$
 $Q_{hum} = ?$

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■ Humidifying (adding water - injection)



Adding water (100% evaporation)

$$\delta = \frac{\Delta h}{\Delta x} = \frac{h_2 - h_1}{x_2 - x_1} = h_w$$

Enthalpy of water

$$h_w = c_w t = 4.187t = 0 \text{ to } 420 \quad [\text{kJ/kg}]$$

Direction of the process

$$\delta = h_w \cong 0 \quad [\text{kJ/g}]$$

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■ Humidifying (adding water)



Adiabatic cooling (air washer)

$$t_{w1} = t_{w2}$$

$$Q = 0$$

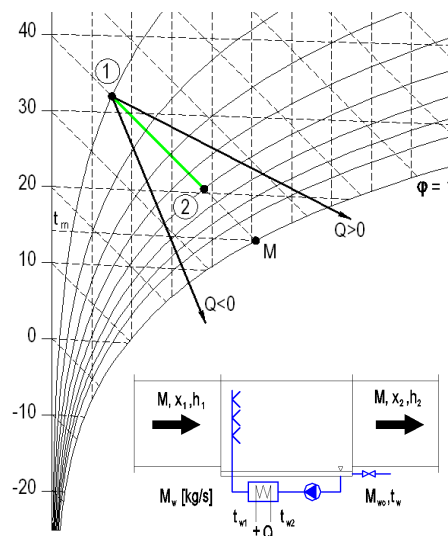
$$\delta = c_w t_w = 4.187t_w \cong 0 \quad \text{kJ/g}$$

Polytropic change

$$t_{w1} \neq t_{w2}$$

$$Q > 0 \quad \dots \delta > 0$$

$$Q < 0 \quad \dots \delta < 0$$



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■ Humidifying (adding water)



Legionella

Legionella acquired its name after a July, 1976 outbreak of a then-unknown "mystery disease" sickened 221 persons, causing 34 deaths. The outbreak was first noticed among people attending a convention of the American Legion - an association of U.S. military veterans. The convention in question occurred in Philadelphia during the U.S. Bicentennial year. This epidemic among U.S. war veterans, occurring in the same city as – and within days of the 200th anniversary of – the signing of the Declaration of Independence, was widely publicized and caused great concern in the United States. On January 18, 1977 the causative agent was identified as a previously unknown bacterium (from air-conditioning) subsequently named *Legionella*. See Legionnaires' Disease for full details.

<http://en.wikipedia.org/wiki/Legionella>

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■ Humidifying (adding water)



Efficiency of air washer

$$\eta = \frac{X_2 - X_1}{X_{wb} - X_1}$$

Example 5:

$$\eta = 60 \%$$

$$t_1 = 27 \text{ }^\circ\text{C}$$

$$\phi_1 = 40 \%$$

$$x_2 = ?$$

$$\phi_2 = ?$$

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■ Dehumidifying



Cooling of air - vapor condensation

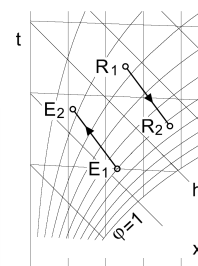
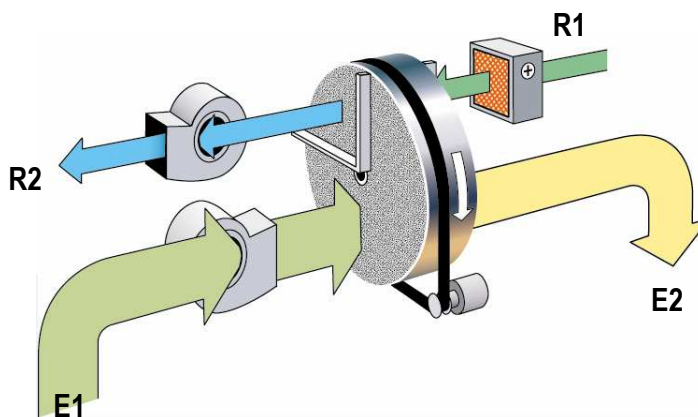
➤ $t_c < t_{dp}$

Adsorption dehumidifying

- rotary heat recovery exchangers
- rotor with **hygroscopic material** as silica gel or activated alumina
- the sorbent material contains a vast number of microscopic pores where water is absorbed
- moisture is condensed and held on the surface of the material
- the adsorbent material can be **reactivated by heat** - regeneration

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■ Dehumidifying



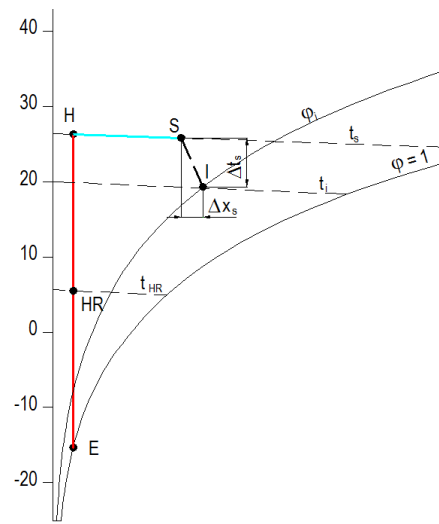
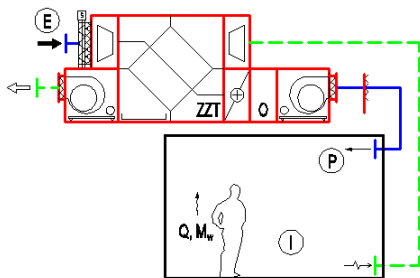
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■ Example of winter process



Single duct air system

- heat recovery
- heater
- humidifying (steam)
- conditioning in the room



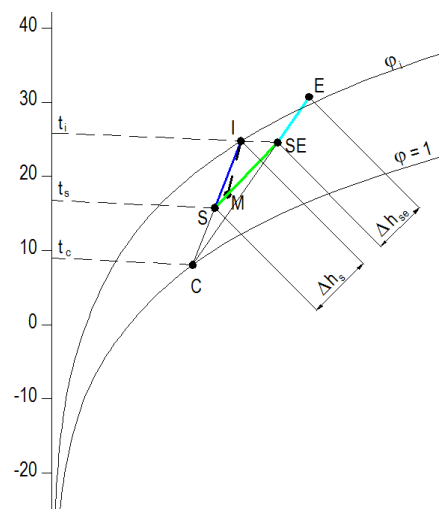
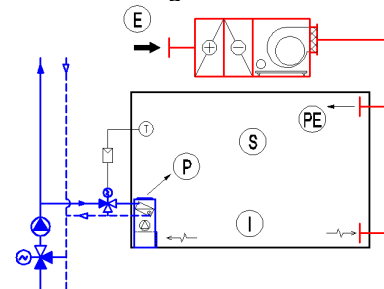
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■ Example of summer process



Fan coil units

- outdoor air cooling
- recirculation of air
- mixing in the room
- conditioning in the room



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**Thank you for your
attention**

